

Beam deflections made easy

Mr A. N. Beal, of Leeds, has sent us a note proposing a simple procedure for the approximate hand calculation of deflections in steel beams. While his contribution has proved too long to be included in Verulam in its entirety, a shortened version may interest many readers. Mr Beal points out that, while the calculation by hand of bending stresses in a beam is not usually difficult, the calculation of deflections may be much more laborious. Since it is not usually necessary to know deflections to any great degree of accuracy (within 10% probably being adequate), the following approach is suggested.

The case of a simply supported beam supporting a uniform load illustrates the approach.

If we take the deflection formula ($\Delta = 5WL^3/384EI$) and express it in terms of the bending moment ($M = WL/8$), it becomes $\Delta = 5ML^3/48EI$.

Now for a steel beam the elastic bending stress $f_{bt} = M/Z$, where $Z = 2I/D$, giving $f_{bt} = MD/2I$. (Z is the elastic modulus, I the moment of inertia and D the section overall depth.)

Inserting this into the deflection formula gives $\Delta = 5 f_{bt}L^3/24ED$. With an E of 210kN/mm³ this becomes:

$$\Delta \text{ (mm)} = 0.992 f_{bt}L^2/D \dots (1)$$

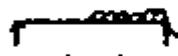
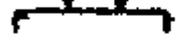
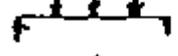
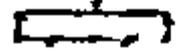
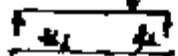
Here f_{bt} , L and D are in their usual units of N/mm², m and mm respectively.

For all practical purposes the formula

$$\Delta = f_{bt}L^2/D \dots (2)$$

is convenient to use, easy to remember and exact to within 1%.

Mr Beal then goes on to consider other load distributions, similarly relating the central deflection Δ to the extreme fibre stress f_{bt} giving results shown in the first column of results in Table 1. The second column gives values for fixed-ended beams, which Mr Beal suggests may be applied to the estimation of deflections in continuous beams. Finally, Mr Beal illustrates how his procedure may be used for complex loadings by deriving the deflection of a simply supported beam loaded as shown in Fig 1:

| Loading | Deflection (mm) | |
|---|-------------------|-------------------|
| | Simply supported | Fixed-ended |
|  | $0.99f_{bt}L^2/D$ | $0.60f_{bt}L^2/D$ |
|  | $0.95f_{bt}L^2/D$ | $0.56f_{bt}L^2/D$ |
|  | $0.89f_{bt}L^2/D$ | $0.51f_{bt}L^2/D$ |
|  | $1.01f_{bt}L^2/D$ | $0.66f_{bt}L^2/D$ |
|  | $0.94f_{bt}L^2/D$ | $0.53f_{bt}L^2/D$ |
|  | $0.79f_{bt}L^2/D$ | $0.40f_{bt}L^2/D$ |
|  | $0.74f_{bt}L^2/D$ | $0.38f_{bt}L^2/D$ |

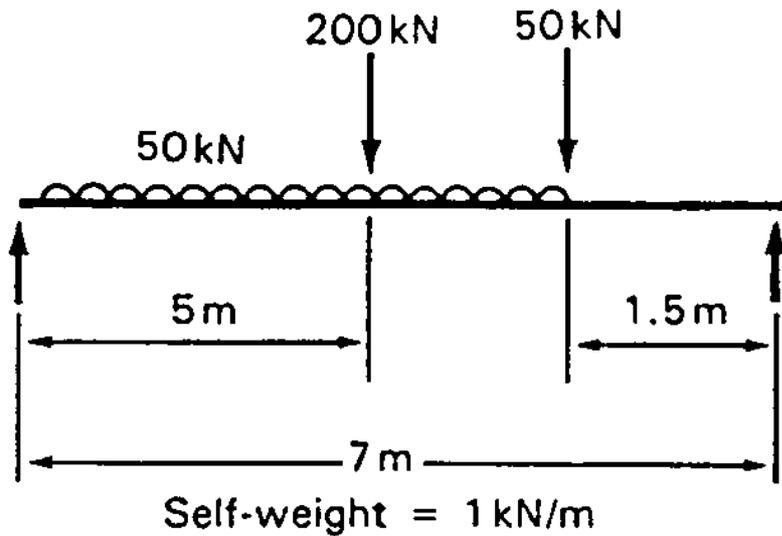


Fig. 1

Central bending moment calculated as 444.3kNm.
 For the beam section, $Z = 2474\text{cm}^3$, $D = 539.5\text{mm}$, giving

$$f_{bt} = 179.6 \text{ N/mm}^2.$$

Simple approximate deflection, using eq. (2) is

$$\Delta_{APP} = 179.6 \times 7^2 / 539.5 = 16.3\text{mm} = L/429 \text{ OK.}$$

For a more exact estimate, considering that most of the moment is generated by the central point load, we might take a coefficient nearer to the point load value of 0.8 (say, 0.85), giving

$$\Delta = 0.85 f_{bt} \times L^2 / D = 13.9\text{mm}$$

For comparison, a precise computer analysis of the same beam gave a deflection of 13.8mm.

Therefore, for most practical purposes, we need to remember only four simple formulae for the deflections of simply supported or continuous steel beams, as shown in Table 2.

Table 2 Deflection

| | Spread load | Point load |
|------------------|------------------|------------------|
| Simply-supported | $f_{bt}L^2/D$ | $0.8f_{bt}L^2/D$ |
| Fixed-ended | $0.6f_{bt}L^2/D$ | $0.4f_{bt}L^2/D$ |

Δ = deflection (mm)

f_{bt} = stress under maximum sagging moment (N/mm²)

L = span (m)

D = depth (mm)

Not only do these formulae make life easy for simple uniform and point loads - they mean that the deflection under more complex load patterns can be calculated without difficulty. They are also particularly suitable for 'back of envelope' checks on computer designs. Best of all, they are easy to remember.

Any takers?